

Computational Fluid Dynamic Modeling of a Downdraft Wood-Fired Furnace

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 - Wood Combustion & Emissions
 - The Problem
 - Addressing the Problem
- 2 The Model
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- 4 Results and Numerical Difficulties
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Motivation

- Heating your home with wood: OWHH (outdoor wood-fired hydronic heater)

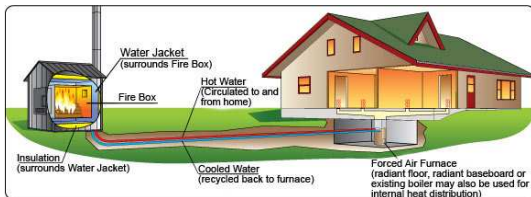


Figure: Typical OWHH Configuration

- All large combustors regulated by the EPA based on National Ambient Air Quality Standards for six air pollutants, three of which are of interest to wood combustion:
- EPA recently adopted a voluntary certification program to curb *PM* emissions from OWHH's

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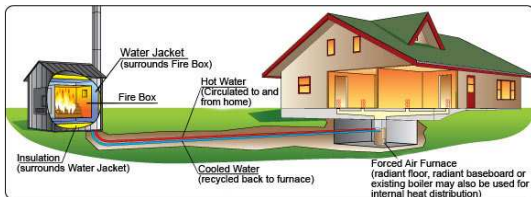


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How Does Wood Burn?

- Wood is a biomass fuel composed of the 4 elements: CHON.
- Wood Combustion proceeds through 4 distinct but overlapping stages:

Stages

- 1 Heating and Drying
 - 2 Pyrolysis and Devolatilization
 - 3 Flaming Combustion
 - 4 Char Oxidation
- Governing processes: chemistry, heat transfer, fluid dynamics.

Emissions

- CO is an intermediate species produced during flaming combustion oxidized to CO_2 .
- Two types of PM emissions: black carbon (i.e. soot, originates in the flame), and brown carbon (organic, originates in pyrolysis).
- CO and PM are emissions due to incomplete combustion - only an 'emission' if they escape the flame.

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Criteria for Complete Combustion

- 1 Time
- 2 Temperature
- 3 Turbulence (mixing)

The Aspen

- Downdraft Unit

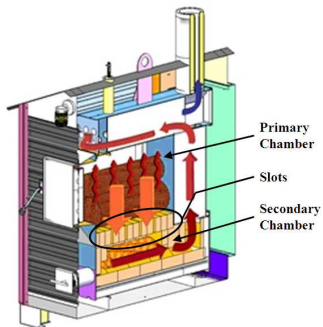


Figure: Schematic of the Aspen's operation

The Problem(s)

- Good *PM* emission levels: 0.27 lbs/million BTU output (meets Phase II EPA certification limit of 0.32).
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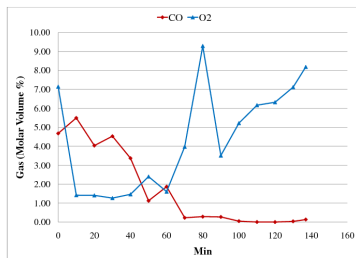
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General

Can we develop a mathematical model that will predict wood combustion emissions?

A Problem of Fluid Dynamics

- A full description of the solid combustion process in an OWHH is not yet practical.
- Narrow the scope: peak *PM* emissions; correlate with peak *CO* emissions
- Model flaming combustion of pyrolysis gas during peak *CO* production (model can therefore be steady state - snapshot).



Governing Equations Summary

Equation Name	Equation
Continuity	$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_j) = 0$
Momentum	$\frac{\partial}{\partial t} (\rho u_i) + \frac{\partial}{\partial x_j} (\rho u_i u_j) = -\frac{\partial P}{\partial x_i} + \frac{\partial \tau_{ij}}{\partial x_j} + \rho g_i$
Energy	$\frac{\partial}{\partial t} (\rho h) + \frac{\partial}{\partial x_j} (\rho u_j h) = \frac{\partial P}{\partial t} + \frac{\partial}{\partial x_j} \left(\frac{k}{C_p} \frac{\partial h}{\partial x_j} - \rho u_j' h' \right) - \frac{\partial \dot{q}_j^R}{\partial x_j}$
Mixture Fraction	$\frac{\partial}{\partial t} (\rho Z) + \frac{\partial}{\partial x_j} (\rho u_j Z) = \frac{\partial}{\partial x_j} \left(\rho D \frac{\partial Z}{\partial x_j} \right)$

Simplifications and Approximations

- Need to solve these equations in a 3-D domain, but turbulence is involved - DNS is too expensive.
- Time average equations to remove 'turbulent fluctuating components' - introduce a turbulence 'closure model' ($k-\epsilon$).
- Also need a radiation transport equation.
- Now we have **7 Parital Differential Equations to solve in 3-D!**

Finite Difference Method

Differential form of a generic conservation equation

$$\frac{\partial(\rho u_j \phi)}{\partial x_j} = \frac{\partial}{\partial x_j} \left(\Gamma \frac{\partial \phi}{\partial x_j} \right) + q_\phi \quad (1)$$

- The idea behind finite difference approximation is borrowed directly from the definition of a derivative:

$$\left(\frac{\partial \phi}{\partial x} \right)_{x_i} = \lim_{\Delta x \rightarrow 0} \frac{\phi(x_i + \Delta x) - \phi(x_i)}{\Delta x} \quad (2)$$

- Extrapolate: replace the partial derivatives by approximations, resulting in one algebraic equation per grid node
- Disadvantages: conservation is not enforced unless special care is taken; method is restricted to simple geometries.

Finite Volume Method

Integral form of a generic conservation equation

$$\int_S \rho \phi \mathbf{v} \cdot \mathbf{n} dS = \int_S \Gamma \nabla \phi \cdot \mathbf{n} dS + \int_{\Omega} \mathbf{q} \phi d\Omega \quad (3)$$

Subdivide domain into finite number of small control volumes (CVs) with a grid that defines the control volume boundaries not the computational nodes.

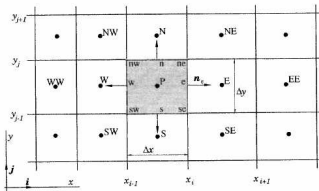


Figure: A typical CV and the notation used for a Cartesian 2D grid (Ferziger)

Finite Volume Method cont....

Implementing the Method

- Approximate surface and volume integrals. Let f be a component of the convective or diffusive flux vector in the direction normal to the CV face.

$$\text{1st Order: } F_e = \int_{S_e} f dS \approx f_e S_e \quad (4)$$

$$\text{2nd Order: } F_e = \int_{S_e} f dS \approx \frac{S_e}{2} (f_{ne} + f_{se}) \quad (5)$$

- Must interpolate to find values at CV surface: *upwind scheme*

$$\phi_e = \phi_P + (x_e - x_P) \left(\frac{\partial \phi}{\partial x} \right)_P + \dots \quad (6)$$

Finite Volume Method cont...

Solution Strategy: linearize algebraic equations, form matrix, iterate.

Advantages:

- Can accommodate any type of grid
- Conservative by construction

Disadvantages:

- Methods of higher than second order accuracy are more difficult to develop in 3D
- Three levels of approximation necessary: integration, differentiation, and interpolation.

How to do this in real life....

- Necessary Components: Equations, Domain, CFD package, Boundary Conditions.
- Need special 'accommodations' for Navier-Stokes Equations
- Software: Fluent

Errors

- 1 Modeling Errors: fuel choice, steady state assumptions, etc.
- 2 Discretization Errors: grid dependency, numerical diffusion.
- 3 Iteration Errors: solution is not fully 'converged'.

My Work

- Flame Structure
- Grid Dependence
- Emissions

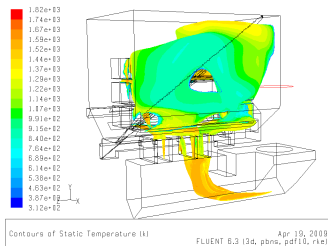


Figure: Grid 1: 4.7 million cells

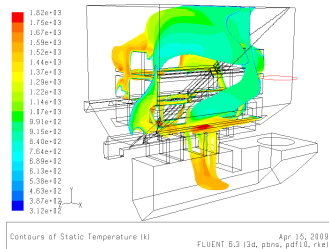


Figure: Grid 2: 5.6 million cells

Issues

Grid Dependent Solution

- Numerical Diffusion
- Computational expense (24 hours, 6GB of RAM, parallel processing)

Under-prediction of stack emissions

- Turbulence modeling?
- Need a transient solution?

Difficult convergence

- buoyancy
- upwind interpolation - second order

Conclusions

- The finite volume method has advantages and disadvantages.
- The simulation cannot be considered a reliable predictor of full furnace performance during peak pyrolysis.
- The simulation can be used to qualitatively understand furnace operation and suggest test scenarios to improve emissions performance and reduce manufacturing costs.

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